

# Transfer number in fine bubble diffused aeration systems

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**Abstract** On the basis of full-scale data from 58 clean water tests performed in 26 activated sludge tanks equipped with fine bubble diffusers and of a theoretical approach, it can be stated that fine bubble aeration systems with total floor coverage arrangement provide higher  $k_L a$  values and the lowest spiral liquid circulation.

An efficiency criterion for oxygen transfer ( $N_T$ ) was defined on the basis of the dimensional analysis. The transfer number  $N_T$  allows us to take account of the impact of vertical liquid circulation movements on oxygen transfer. The values of  $N_T$  calculated from the results of full scale nonsteady-state clean water tests vary from  $5.3 \times 10^{-5}$  to  $9.1 \times 10^{-5}$  and are directly dependent upon the arrangement of air diffusers. It has been shown that the highest transfer numbers corresponded to the total floor coverage arrangement and the average calculated  $N_T$  values is  $7.7 \times 10^{-5}$ , independently of the diffuser density and of the gas velocity, over the ranges studied. The lowest transfer numbers are obtained when the diffusers are located in separate grids, and the transfer number is reduced with increasing air flow rate.

**Keywords** Clean water tests; fine bubble aeration; oxygen transfer

## Nomenclature

- $a$  = specific interfacial area ( $\text{m}^2 \cdot \text{m}^{-3}$ )
- $d_B$  = diameter of air bubbles (m)
- DD = diffuser density (%)
- $D_{O_2}$  = coefficient of oxygen molecular diffusion ( $\text{m}^2 \cdot \text{s}^{-1}$ )
- $g$  = acceleration of gravity ( $\text{m} \cdot \text{s}^{-2}$ )
- $G$  = gas-liquid slip velocity ( $\text{m} \cdot \text{s}^{-1}$ )
- $H_{\text{imm}}$  = depth of submergence of diffusers (m)
- $k_L$  = oxygen transfer coefficient ( $\text{m} \cdot \text{s}^{-1}$ )
- $k_L a$  = volumetric oxygen transfer coefficient ( $\text{h}^{-1}$ )
- $N_T$  = transfer number (–)
- $t_c$  = contact time of the bubble in the liquid (s)
- $U_G$  = superficial gas velocity ( $\text{m} \cdot \text{s}^{-1}$ )
- $U_L$  = superficial liquid velocity ( $\text{m} \cdot \text{s}^{-1}$ )
- $U_t$  = terminal rise velocity ( $\text{m} \cdot \text{s}^{-1}$ )

## Greek letters

- $\varepsilon_G$  = gas holdup (–)
- $\varepsilon_L$  = liquid holdup (–)
- $\mu$  = dynamic viscosity of the liquid ( $\text{kg} \cdot \text{m}^{-1} \cdot \text{s}^{-1}$ )
- $\rho$  = liquid density ( $\text{kg} \cdot \text{m}^{-3}$ )
- $\sigma$  = surface tension of the liquid ( $\text{dyne} \cdot \text{cm}^{-1}$ )

## Introduction

Oxygen transfer in aeration basins is generally characterised by the volumetric oxygen transfer coefficient ( $k_L a$ ). In the case of fine bubble diffused air systems, the main

parameters influencing  $k_L a$  in clean water are the air flow rate (Déronzier *et al.*, 1998; Gillot and Héduit, 1998; Wagner and Pöpel, 1998), the arrangement and density of air diffusers (ASCE, 1994; Wagner and Pöpel, 1998) and the depth of submergence of the diffusers (Jackson and Shen, 1978; Khudenko, 1986; Pöpel and Wagner, 1994).

The effect of these parameters on oxygen transfer, by modifying the bubble size, the hydrodynamics and/or the contact time of the bubbles in the liquid, varies from one site to another. To compare the performance of different systems or to transpose results from one scale to another, dimensional analysis is commonly used (Khudenko, 1986; Fujie, 1997). Two systems characterised by the same dimensionless variables are said to be similar and present the same level of performance (Zlokarnik, 1998).

The purpose of this paper is to define, using dimensional analysis, an oxygen transfer efficiency criterion in fine bubble diffused air systems and to examine its capacity for assessing the impact of the arrangement of diffusers on oxygen transfer. The effect of diffuser density and variations in the air flow rate is also examined.

## Theoretical approach

### Volumetric oxygen transfer coefficient [ $k_L a$ ] in clean water

The volumetric oxygen transfer coefficient [ $k_L a$ ] is defined as the product of the oxygen transfer coefficient [ $k_L$ ] and the specific interfacial area [ $a$ ].

*The oxygen transfer coefficient.* [ $k_L$ ] depends on the size of the bubbles ( $d_B$ ), on the terminal rise velocity ( $U_t$ ), on temperature ( $T$ ) and on the characteristics of the liquid ( $\sigma$ ,  $\mu$ ,  $\rho$ ):  $k_L = f(d_B, U_t, T, \sigma, \mu, \rho \dots)$ .

For bubble diameters of more than 2.5 mm,  $k_L$  can be determined from Higbie's penetration model (1935):

$$k_L = 2 \sqrt{\frac{D_{O_2}}{\pi t_c}} = 1.13 \sqrt{\frac{D_{O_2}}{t_c}} \quad \text{with } t_c = \frac{d_B}{U_t} \quad (1)$$

For bubble diameters in the range of 3 and 4 mm (produced by fine bubble diffused air systems), the terminal rise velocity ( $U_t$ ) remains constant at  $0.24 \text{ m}\cdot\text{s}^{-1}$  in clean water (Stenstrom and Gilbert, 1981). For a given temperature,  $k_L$  can also be expressed as:  $k_L = f(d_B)$ .

*The specific interfacial area* [ $a$ ] depends on the gas holdup and on the bubble size:

$$a = \frac{6}{d_B} \frac{\varepsilon_G}{1 - \varepsilon_G} \quad (2)$$

The gas holdup, defined as the air content in the water, can be determined using the slip model of Wallis (1969):

$$= \frac{U_G}{\varepsilon_G} - \frac{U_L}{\varepsilon_L} \quad (3)$$

and

$$\varepsilon_G + \varepsilon_L = 1 \quad (4)$$

$U_G$  and  $U_L$  are the superficial gas and liquid velocities co-currently in a bubble column.

By combining the Eqs. (3) and (4), the gas holdup is written as follows:

$$\varepsilon_G = \frac{(U_G + U_L + G) \pm \sqrt{(U_G + U_L + G)^2 - 4U_G G}}{2G} \quad (5)$$

The gas-liquid slip velocity ( $G$ ) can be assimilated to the terminal rise velocity :  $G \approx U_t$  (Shal *et al.*, 1982). This is written :  $a = f(d_B, U_G, U_L)$ .

The volumetric oxygen transfer coefficient  $k_L a$ . By combining the Eqs. (1), (2) and (5),  $k_L a$  can be estimated from the following equation :

$$k_L a_T = 1.13 \sqrt{\frac{D_{O_2} U_t}{d_B}} \cdot \frac{6}{d_B} \cdot \frac{(U_G + U_L + G) \pm \sqrt{(U_G + U_L + G)^2 - 4U_G G}}{(-U_G - U_L + G) \mp \sqrt{(U_G + U_L + G)^2 - 4U_G G}} \quad (6)$$

**Parameters influencing oxygen transfer in clean water**

The theoretical approach above (Eq. (6)) makes it possible to calculate the volumetric oxygen transfer coefficient  $k_L a$  in clean water, taking account of the main parameters involved:  $k_L a = f(d_B, U_L, U_G)$ .

In clean water, the *bubble size* ( $d_B$ ) which essentially depends on the type of diffuser and on the pore size (Da Silva-Déronzier, 1994; Hébrard, 1995) determines the surface area (cf. Eq. 2).

The *superficial liquid velocity* ( $U_L$ ) is generated by the air-lift action of the air bubbles rising in the liquid.  $U_L$  depends essentially on the diffuser arrangement (Fujie, 1997). When the diffusers are arranged in a grid occupying only a part of the bottom of the basin (Figure 1a), the liquid driven by the bubbles preferentially moves to those parts of the basin devoid of diffusers, thus creating a rotation of the liquid, increasing the rise velocity of the bubbles and reducing their contact time in the liquid. On the other hand, when the diffusers are uniformly arranged on the floor of the basin (Figure 1b), the liquid circulation movements are smaller and the liquid velocity distinctly reduced.

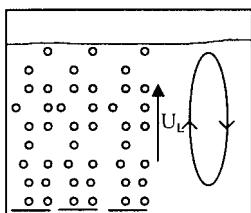
Figure 2 presents (dotted lines), for a bubble diameter set at 3.5 mm (mean diameter generated by fine bubble systems), the influence of the *gas velocity*  $U_G$  on the volumetric oxygen transfer coefficient  $k_L a$  for different vertical velocities of the liquid  $U_L$ , according to the Eq. (6).

For a constant liquid velocity ( $U_L$ ), the volumetric oxygen transfer coefficient  $k_L a$  is an increasing function of the *gas velocity*. It can also be seen that at a given gas velocity a reduction of  $k_L a$  by approximately 50% can be expected when the vertical component of the liquid velocity rises from 0 to 0.20 m.s<sup>-1</sup>.

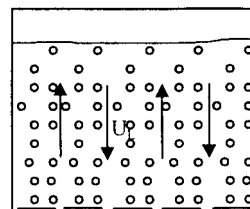
**Dimensional analysis**

Previous studies based on the theory of similarity have evidenced that a dimensionless number, characteristic of the oxygen transfer, can be defined : the *transfer number* ( $N_T$ ) (Zlokarnik, 1978; Roustan, 1996) :

$$N_T = \frac{k_L a}{U_G} \left( \frac{\mu^2}{\rho^2 g} \right)^{1/3} \quad (7)$$



**Figure 1a** Diffusers arranged in grids



**Figure 1b** Total floor coverage arrangement

The transfer number  $N_T$  characterises the oxygen transfer efficiency of an air diffused system. To within the acceleration of gravity ( $g$ ) and the characteristics of the liquid ( $\mu, \rho$ ), the transfer number relates the same physical significance as the oxygen transfer efficiency per metre of immersion.

According to the theoretical approach above,  $N_T$  can be written :  $N_T=f(U_G, U_L, d_B)$ . Assuming that fine bubble diffused air systems (discs or tubes) produce bubbles of the same size, the transfer number essentially becomes a function of the gas and liquid velocities :  $N_T=f(U_G, U_L)$ .

### Materials and method

The results of 58 measurements of oxygen transfer efficiency in clean water carried out over the last decade by the Cemagref in 26 basins of different geometry and volume (water depth under 7 m) are given in Appendix I. These data focus solely on fine bubble diffused air systems (discs or tubes) without any horizontal liquid velocity induced by a mechanical agitation. The oxygen transfer coefficients were measured in clean water according to the nonsteady-state method (Héduit and Racault, 1983; ASCE, 1992; Duchène *et al.*, 1995; ATV, 1996).

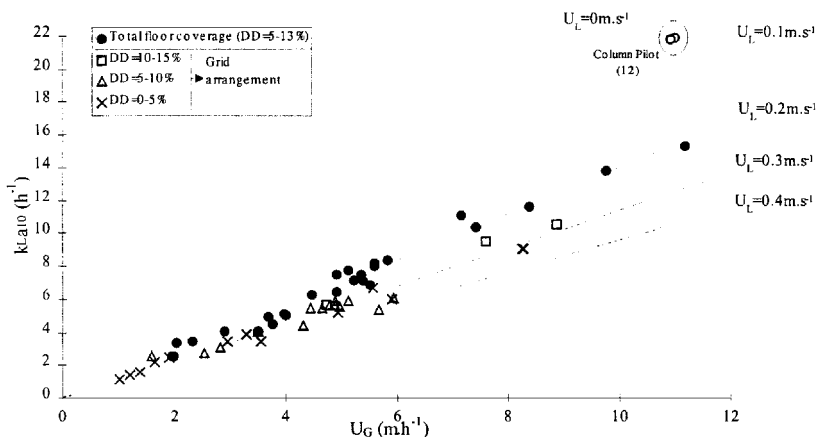
The diffuser density is expressed as the ratio between the total (perforated) area of the membranes and the total surface area of the oxidation tank. The gas velocity is defined as the air flow rate divided by the surface area of the basin, the air flow rate being expressed at a temperature of 0°C and a pressure at mid-height of  $(1013+\Delta P/2)$ hPa (where  $\Delta P$  is the hydrostatic pressure). The transfer number is calculated from the volumetric oxygen transfer coefficients measured, from the gas velocity and from the dynamic viscosity of clean water, according to Eq. (7).

The transfer number was calculated for each site tested and its capacity to take account of the impact of the liquid circulation movements created by the arrangement and density of the diffusers and the variations in air flow was examined.

### Results and discussion

#### Volumetric oxygen transfer coefficient $k_L a$

Figure 2 shows all the experimental volumetric oxygen transfer coefficients and the theoretical curves (dotted lines) of  $k_L a$  taken from Eq. (6) as a function of  $U_G$  and  $U_L$ .



**Figure 2** Influence of the gas and liquid superficial velocities on the volumetric oxygen transfer coefficient at 10°C from different full-scale clean water aeration tests

All the experimental data are characterised by a theoretical liquid velocity ranged from 0.1 to 0.4 m.s<sup>-1</sup>.

The total floor coverage arrangement corresponds to the highest  $k_L a$  values and to the lowest circulation velocities ( $U_L \approx 0.15$  to 0.25 m.s<sup>-1</sup>), irrespective of the gas velocities and of the diffuser density.

When the diffusers are arranged in separate grids, the liquid circulation velocities are higher (between 0.25 and 0.4 m.s<sup>-1</sup>), especially when the diffuser density is below 10%.

The pilot installation (site 12) is characterised by a higher  $k_L a$  value, explained by a high diffuser density (35%) and a low liquid velocity (below 0.1 m.s<sup>-1</sup>).

### The transfer number ( $N_T$ )

Irrespective of the basin geometry (cylinder, channel, parallelepiped) and the volume of liquid (from 18 m<sup>3</sup> to 5000 m<sup>3</sup>), the transfer number in full scale plants is ranged between  $5.3 \times 10^{-5}$  to  $9.1 \times 10^{-5}$  (cf. Appendix I).

The pilot installation (site 12) is characterised by high transfer numbers ( $11 \times 10^{-5}$ ), which evidences that the hydrodynamics of this system are not comparable to those of full-size aeration tanks and that any extrapolation of the results obtained in the pilot to the performance to be expected at full-scale plants must be considered with circumspection.

Figure 3 shows the influence of the diffuser layout on the transfer number  $N_T$ , for a diffuser density comprised between 7 and 10% and a gas velocity of approximately 5 m.h<sup>-1</sup>.

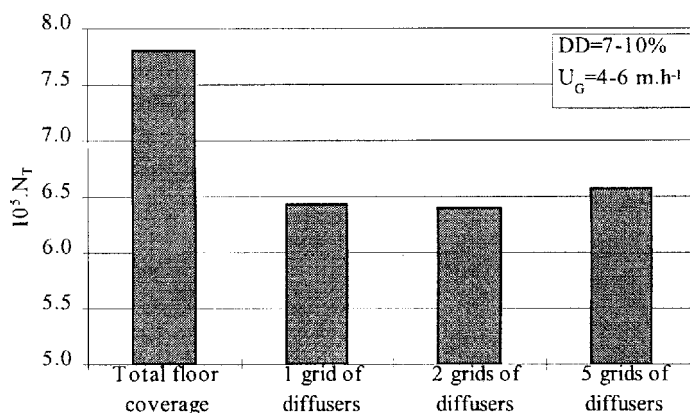
The transfer number is on average  $7.8 \times 10^{-5}$  when the diffusers are located uniformly on the bottom of the tank and between  $6.4 \times 10^{-5}$  and  $6.6 \times 10^{-5}$  when the diffusers are arranged in separate grids.

The transfer number  $N_T$  thus allows us to translate the influence of the *arrangement* of the diffusers on the transfer efficiency through the vertical liquid velocity  $U_L$  induced between the grids of diffusers.

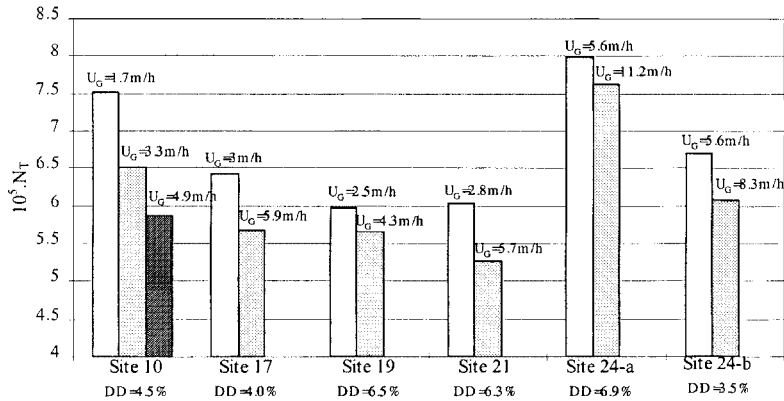
*Full-floor coverage systems.* The highest transfer numbers (cf. Appendix I) correspond to the total floor coverage arrangement tested : the average of the 26 values calculated is  $N_T = 7.7 \times 10^{-5}$  (n=26, standard deviation=0.6), with only two  $N_T$  values under  $7.0 \times 10^{-5}$ .

Over the ranges studied, the effect of an increase in gas velocity and of a variation in diffuser density on the transfer number is not noticeable (cf. Appendix I).

The liquid circulation movements generated in this configuration occur between the diffusers and result therefore in low vertical liquid velocities, having little effect on the transfer number.



**Figure 3** Influence of the diffuser layout on the transfer number (Site no. 1, 3, 8, 9, 14, 24, 25 and 26)



**Figure 4** Influence of the gas velocity on the transfer number (grid arrangement)

*Basins with separate grids of diffusers.* The average of the transfer numbers when the diffusers are located in separate grids is lower than that of the transfer numbers for basins with a total floor coverage arrangement:  $N_T = 6.4 \times 10^{-5}$  ( $n=30$ , standard deviation=0.7).

As for full-floor coverage systems, variations in the diffuser density over a range from 3% to 15% do not appear to influence the transfer number (cf. Appendix I).

On the other hand, Figure 4 shows the effect of an increase in the gas velocity on the transfer number for 5 different wastewater treatment plants, where the diffusers are arranged in separate grids (diffuser density between 3.5% and 7%).

An increase in the air flow rate per surface area unit of the basin results in a decrease of  $N_T$ .

The influence of variations in the air flow rate on the transfer number  $N_T$  is thus more noticeable when the grids of diffusers are spaced out, namely when the liquid circulation rates occur between the grids are the greatest (see Figure 4), than in full-floor coverage systems where the vertical liquid velocities are induced between the diffusers.

## Conclusions

The volumetric global oxygen transfer coefficients [ $k_L a$ ] determined in 58 clean water tests were depicted according to the gas velocity measurement and the vertical liquid velocity, determined by a theoretical approach.

This evidenced that the highest [ $k_L a$ ] values, and the lowest vertical liquid circulation velocities are found for full-floor coverage systems, irrespective of the gas velocity and diffuser density.

The dimensional analysis allowed to define a dimensionless number, characteristic of the oxygen transfer. This efficiency criterion, written as  $N_T$ , is expressed according to the volumetric oxygen transfer coefficient, the gas velocity and the liquid characteristics ( $\rho$ ,  $\mu$ ).

The experimental values of  $N_T$  were then calculated from the results of clean water tests. The values of  $N_T$  vary from  $5.3 \times 10^{-5}$  to  $9.1 \times 10^{-5}$  and are directly dependent on the arrangement of the diffusers.

- The highest transfer numbers ( $N_T = 7.7 \times 10^{-5}$ ,  $n=26$ , standard deviation=0.6) correspond to total floor coverage arrangements of diffusers (low vertical liquid circulation between the diffusers).
- Smaller transfer numbers ( $N_T = 6.4 \times 10^{-5}$ ,  $n=30$ , standard deviation=0.7) are obtained when the diffusers are located in separate grids (high vertical liquid circulation between the grids of diffusers).

- Over the range studied, the diffuser density, irrespective of the diffuser layout, has little influence on the transfer number (3% to 15%).
- When the diffusers are located in separate grids, an increase in the air flow rate, amplifying the liquid movements between the grids of diffusers, results in a drop in the transfer number.

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## Appendix I

Site	Basin geometry			Diffuser				
	Geometry	Submergence (m)	Volume (m <sup>3</sup> )	Arrangement	Type	Density (%)	U <sub>G</sub> (m.h <sup>-1</sup> )	IO <sup>2</sup> .N <sub>T</sub> (-)
1	Cylinder	5.2	240	total floor coverage	Disc	8.9	3.5	6.5
		2.2	108			8.9	4.0	7.0
		5.3	243			8.9	5.5	7.0
2	Cylinder	5.5	638	total floor coverage	Disc	4.9	2.0	7.4
		2.3	282			4.9	2.0	7.1
		5.2	597			4.9	3.8	6.7
3	Cylinder	5.9	475	total floor coverage	Disc	7.5	4.5	7.8
		5.9	475			7.5	5.8	8.0
		2.9	243			7.5	5.4	7.7
		2.9	243			7.5	7.4	7.7
4	Cylinder	3.5	347	total floor coverage	Disc	5.3	3.7	7.4
5	Cylinder	3.5	346	1 grid	Disc	6.5	5.9	5.7
6	Cylinder	4.0	397	3 grids	Disc	14.3	7.6	7.0
		4.0	396	2 grids		10.6	8.9	6.6
		4.0	396	1 grid		3.7	3.6	5.3
7	Cylinder	3.8	318	1 grid	Disc	8.4	4.9	6.8
8	Cylinder	4.2	757	2 grids	Disc	7.0	4.9	6.4
		4.2	757			7.0	4.8	6.5
		5.1	757			total floor coverage	Disc	6.8
10	Cylinder	5.0	1327	8 grids	Disc	4.5	4.9	5.9
		5.0	1318			4.5	3.3	6.5
		5.0	1318			4.5	1.7	7.5
11	Cylinder	4.6	751	total floor coverage	Tube	12.7	8.4	7.7
		4.6	751			12.7	7.2	8.6
		4.6	750			12.7	5.4	7.4
		4.9	795		Tube	6.1	4.9	7.3
		4.9	795			6.1	4.0	7.1
		4.9	795			6.1	2.9	7.8
12	Cylinder	1.3	0.2	pilot	Disc	34.4	11.0	11.1
		2.7	0.4			34.4	10.9	11.1
13	Cylinder	4.8	850	3 grids	Disc	9.6	5.0	6.2
14	Cylinder	4.6	1451	5 grids	Disc	9.0	4.6	6.6
15	Channel	2.5	1364	10 grids	Disc	5.9	1.6	8.8
		2.5	1364	5 grids		3.0	1.4	6.5
16	Channel	4.1	1572	4 grids	Plate	4.5	1.2	6.3
17	Channel	4.2	2070	12 grids	Disc	4.0	3.0	6.4
		4.2	2070	12 grids		4.0	5.9	5.7
18	Channel	2.3	117	2 grids	Tube	5.5	4.4	6.9
19	Channel	4.7	1364	8 grids	Disc	6.5	4.3	5.6
		4.7	1348	8 grids		6.5	2.5	6.0
20	Channel	4.7	1034	4 grids	Disc	4.8	1.9	7.3
21	Channel	3.3	579		Tube	4.9	2.8	6.0
		3.3	579			4.9	5.7	5.3
22	Channel	3.1	216	2 grids	Tube	6.7	3.5	6.5
23	Parallelpiped	7.0	5102	11 grids	Tube	11.4	4.7	6.6
		7.0	2913	4 grids		1.9	1.0	6.0
24	Parallelpiped	3.5	18		Tube	6.9	5.6	8.0
		3.5	18			6.9	5.6	8.2
		3.5	18			6.9	11.2	7.6
		3.5	18			3.5	5.6	6.7
		3.5	18			3.5	8.3	6.1
		3.5	18			3.5	8.3	6.1
25	3/4 cylindrical	4.6	1428	total floor coverage	Disc	9.0	2.0	9.1
		3.2	1012			9.0	2.3	8.3
		4.6	1434			9.0	4.9	8.4
		4.0	1250			9.0	5.1	8.4
		4.6	1443			8.9	9.8	7.9
26	Truncated cylinder	4.8	167	1 grid along the wall	Disc	9.9	5.1	6.4