

## 1 **Experimental and computational study of oil jet in crossflow**

2 Cosan Daskiran<sup>1</sup>, Fangda Cui<sup>1</sup>, Lin Zhao<sup>2§</sup>, Scott A. Socolofsky<sup>3</sup>, Kenneth Lee<sup>4</sup>, Michel C. Boufadel<sup>1\*</sup>

3 <sup>1</sup>*Center for Natural Resources, Civil and Environmental Engineering Department, New Jersey Institute*  
4 *of Technology, Newark, NJ 07102*

5 <sup>2</sup>*ExxonMobil Upstream Research Company, Houston, TX 77389*

6 <sup>3</sup>*Zachry Department of Civil Engineering, Texas A&M University, College Station, TX 77843*

7 <sup>4</sup>*Bedford Institute of Oceanography, Department of Fisheries and Oceans, Dartmouth, Canada*

8 \*Corresponding author: Michel C. Boufadel ([boufadel@gmail.com](mailto:boufadel@gmail.com); <http://nrdp.njit.edu>)

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10 Technology

### 11 **Abstract**

12 During undersea oil blowout in crossflow conditions, the oil droplets entrained horizontally which  
13 increased the residence time of droplets in the water column. Knowledge of the trajectory of an oil  
14 plume is important for predicting the pathways of hydrocarbons and to devise countermeasures. We  
15 conducted large-scale experiments in the Ohmsett tank where we released oil from a one-inch vertical  
16 orifice that was towed to produce the behavior of a jet in crossflow. The average oil velocity at the  
17 orifice was 1.36 m/s and the crossflow velocity was around 0.27 m/s which resulted in a jet-to-crossflow  
18 velocity ratio of 5.0. The results were simulated numerically using the Large Eddy Simulation (LES)  
19 turbulence model and the mixture multiphase model within the open-source software OpenFOAM. The  
20 instruments including ADVs, LISSTs, shadowgraph cameras, holographic camera, and fluorometers  
21 were employed. The oil jet released from the nozzle started to meander in the vertical direction most  
22 probably due to weak crossflow. The trajectory and meandering behavior of the oil jet, wavy pattern  
23 along the leading edge of the jet and column breakup observed in the experiments were captured well  
24 with the numerical simulation. The surface breakup just above the orifice created ligaments and droplets  
25 downstream of the jet. Larger oil droplets were observed near the upper boundary of the plume due to  
26 their higher buoyancy while the smaller droplets were suspended in the water column and they were  
27 entrained by water crossflow. This work reveals that different size of droplets determines the overall  
28 shape of plumes mostly the upper and lower boundaries of the plume.  
29

## 30 **1 Introduction**

31 The Deepwater Horizon oil spill in 2010 is recognized to be the worst environmental disaster in  
32 U.S. history with an estimated oil release of 4.9 to 6.2 million barrels (Griffiths 2012), and highlighted  
33 the need for a better understanding of underwater oil jets and plumes. A comprehensive review for  
34 droplets, hydrodynamics and chemistry of multiphase jets/plumes including recent numerical and  
35 experimental studies was conducted by Boufadel et al. (2020). Due to horizontal currents in the ocean,  
36 the released jet could bend horizontally causing the oil droplets and gas bubbles to separate from it. In  
37 spite of the importance of this behavior on the trajectory and fate of oil, there is little understanding of  
38 the residence time of the droplets within the plume, which is likely to be increased due to turbulence.  
39 For this reason, we report herein the results of an experimental release of oil at 40 liter/min from a one-  
40 inch pipe in the Ohmsett tank in New Jersey. The experimental results were simulated using high-  
41 fidelity (20 million cells) large eddy simulation turbulence model and mixture multiphase model.

42 The droplet size is crucial to estimate the trajectory and the residence time of the droplets in the  
43 water column which becomes important for the oil spill response team to set the countermeasures. Based  
44 on the estimated trajectory of oil droplets, the region to apply underwater dispersant can be determined  
45 more precisely. The dispersant is used to decrease the interfacial tension between oil and water and  
46 creates smaller droplets. The decrease in the droplet size decreases the rise velocity of droplets and  
47 increases their residence time in the water column which increases the biodegradation and dissolution  
48 rate of oil droplets into water. In addition to the underwater dispersant application, estimating the  
49 location of the oil slick before the droplets reach the water surface provides additional time for the  
50 response team to get prepared for the countermeasures. The density stratification and horizontal streams  
51 in the water column have a major role on oil transport. The role of waves on the transport of surfaced  
52 oil droplets was investigated in our recent work (Cui et al. 2020a, Cui et al. 2020b). In this work, we  
53 focused on the influence of horizontal crossflows on oil trajectory. Another crucial parameter for  
54 numerical models to estimate oil trajectory accurately is the initial droplet size which was considered  
55 to be generated through the shear among water and oil and resulting Kelvin-Helmholtz instabilities  
56 above the orifice (within a few diameters) (Funada et al. 2004, Bo et al. 2011, Xue and Katz 2019).

57 Daskiran et al. (2020a) investigated the formation of ligaments and droplets in the shear breakup (also  
58 called primary breakup) zone.

59 Jets penetrate into the crossflow based on their initial momentum and starts bending in the crossflow  
60 direction. The flow field and vortical structures have been characterized through numerical and  
61 experimental studies from small-scale jets (Yuan et al. 1999, Gopalan et al. 2004, Muppidi and Mahesh  
62 2007, Mahesh 2013, Behzad et al. 2016). Yuan et al. (1999) observed rollers in a spanwise direction  
63 near the leading and trailing edges of the jet due to the Kelvin Helmholtz (K-H) instabilities which  
64 occur at the phase interface due to the velocity difference. Muppidi and Mahesh (2007) investigated  
65 turbulent kinetic energy budgets through direct numerical simulations. In the near field of the jet, they  
66 observed peak kinetic energy production near the leading edge of the jet and peak energy dissipation  
67 rate near the trailing edge of the jet. Gopalan et al. (2004) characterized the jet in crossflow based on  
68 the velocity ratio,  $r$ , the ratio of jet velocity to crossflow velocity. They reported a semi-cylindrical  
69 vortical layer with a reverse flow behind the jet for the velocity ratio smaller than 2.0. They did not  
70 observe the reverse flow for higher velocity ratios. Instead, they observed vortical structures behind the  
71 jet similar to Karman vortex street. Behzad et al. (2016) investigated the surface breakup and the  
72 formation of ligaments and droplets from the jet shear layer due to the azimuthal instabilities along the  
73 jet periphery. Many studies for the jet in crossflow were performed as single-phase or multiphase such  
74 as liquid jet in gaseous crossflow. Murphy et al. (2016) investigated oil jet in crossflow experimentally  
75 by towing the pipe and investigating the water velocity (and oil velocity) using Particle Image  
76 velocimetry (PIV). They revealed and quantified the vertical velocity induced by a counter-rotating  
77 vortex pair (CVP) beneath the plume. Recently, Daskiran et al. (2020b, 2021) conducted a large eddy  
78 simulation and reported detailed hydrodynamics of jet in crossflow and the role of CVP on oil  
79 distribution and mixing. Using the hydrodynamics from the LES, Daskiran et al. (2020c) carried out  
80 Lagrangian particle tracking simulations using their in-house code “NEMO3D” (Cui et al. 2018, Cui et  
81 al. 2020a). They investigated the transport of droplets in various sizes and trapping of small droplets  
82 (i.e. a few hundred microns) inside the CVP.

83 The objective of this work is to compare large-scale experiments of the oil jet in crossflow to

84 numerical simulations. To the best knowledge of the authors, there are not sufficient number of studies  
 85 for large-scale multiphase liquid-liquid jet flows in crossflows. This work will aid in estimating the oil  
 86 trajectory and overall shape of plumes mostly the upper and lower boundaries of the plume in crossflow.

## 87 2 Experimental and simulation Setup

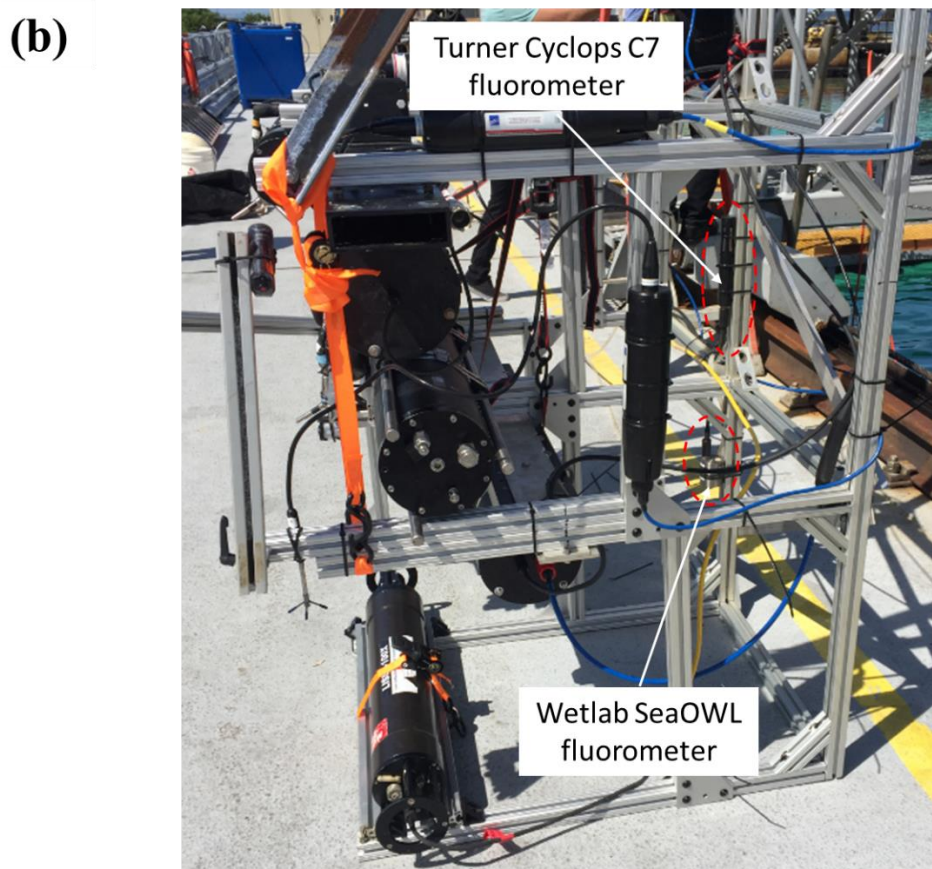
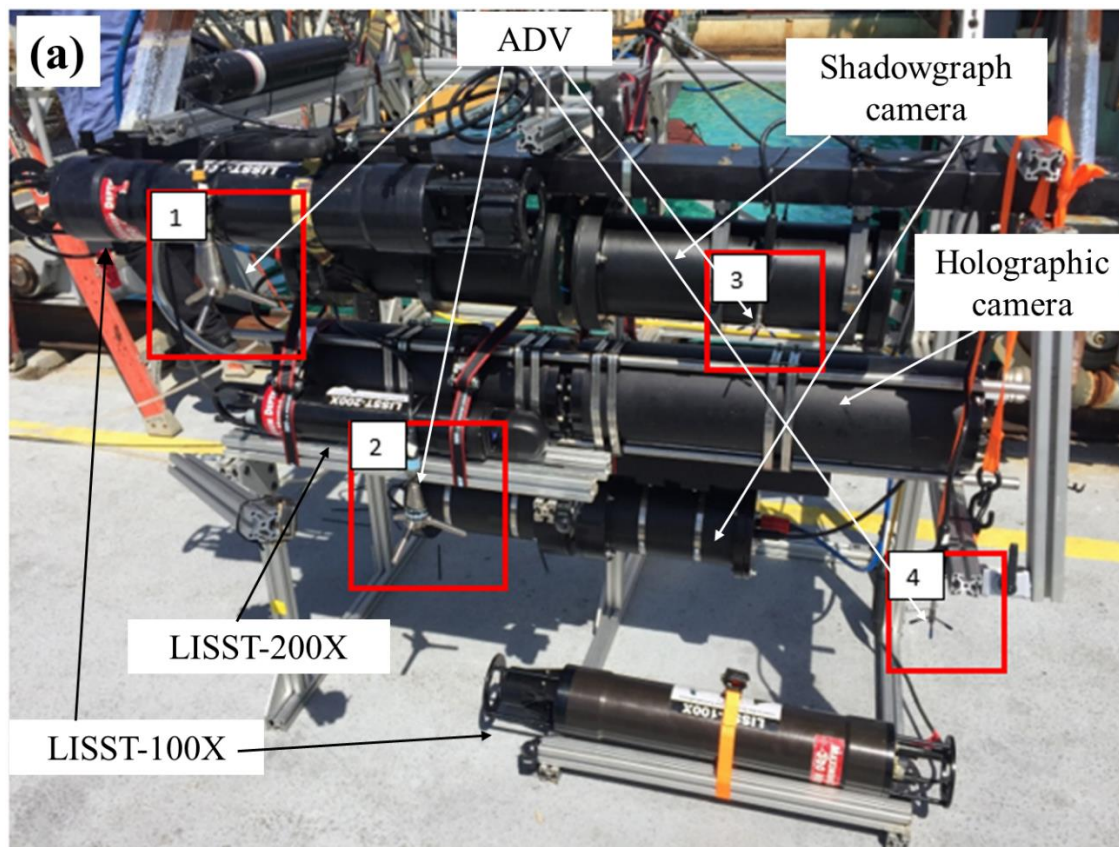
88 The experiments of the jet in crossflow were conducted at Ohmsett wave tank facility which has  
 89 203 m length, 20 m width and 2.4 m depth. The oil type used in the experiments was Fuel oil #2. The  
 90 properties of the oil and seawater were provided in Table 1. The parameter  $D$  in Table 1 represents the  
 91 pipe diameter. In addition to the experiments, large eddy simulation was performed for the same flow  
 92 conditions.

93 Table 1. The properties of oil and seawater used in the experiment and simulation.

$\rho_{oil}$	$\rho_{water} [kg/m^3]$	$\mu_{oil} [cP]$	$\mu_{water} [cP]$	$\sigma [mN/m]$	$D [mm]$
840	1025	4	1.1	25.6	25

94

95 In the experiments, the pipe and the instruments were towed to mimic the crossflow current. Several  
 96 instruments were installed on the metallic frames downstream of the orifice to measure flow velocity,  
 97 oil droplet size and oil concentration in water. Figure 1 shows the snapshots of the metallic frame with  
 98 different instruments installed. Four Acoustic Doppler Velocimetry (ADV) were installed to measure  
 99 velocity in three-dimensions as a function of time at specified measurement points. Three LISSTs  
 100 (Laser in-situ scattering and transmissometry), two LISST-100X and one LISST-200X, were installed  
 101 to reveal the oil droplet size distribution (DSD) in the range of 2.5-500 micron. Two shadowgraph  
 102 cameras were employed to capture the droplets larger than 500 microns. The shadowgraph cameras  
 103 used herein are telecentric and consist of two cylindrical parts with a gap between to allow the oil  
 104 droplets pass through. One side of the camera is the light source while the other side includes the  
 105 camera. The images of droplets were taken while they are passing through the gap with a frequency of  
 106 10 Hz. The holographic camera was also installed to capture images of droplets. Two fluorometers  
 107 named “Wetlab SeaOWL” and “Turner Cyclops C7” were used to measure dissolved oil concentration  
 108 in water. The LISSTs and the cameras were installed carefully to measure the droplet size across the  
 109 plume cross-section particularly near the upper and lower boundaries of the plume.



110

111 Figure 1. Location and setup of the instruments mounted on metallic frames. Four ADVs, three LISSTs,  
 112 two shadowgraph cameras, one holographic camera and two fluorometers were used.

113 In the simulation, a rectangular computational domain was employed. The oil was injected in the  
114 vertical direction while crossflow was moving in the horizontal direction. The pipe length was  $20D$  and  
115 the vertical distance between the orifice and water free surface was roughly  $70D$ . The pipe was placed  
116  $160D$  away from the water channel inlet. The distance between the pipe and channel outlet was  $240D$   
117 to allow the oil jet to bend in the crossflow direction. At the water inlet, a crossflow velocity of  $0.27$   
118 m/s was used. At the pipe inlet, oil had an average velocity of  $1.36$  m/s which corresponds to an oil  
119 flow rate of  $40$  liters/min. At the outlet, a zero-gradient condition was applied for the velocity in the  
120 direction normal to the surface. Zero-shear wall and no-slip wall boundary conditions were employed  
121 for the side boundaries of the computational domain and the pipe wall, respectively. The number of  
122 cells used in the simulation was around  $20$  million. The cell size was refined carefully near the orifice  
123 and in the region occupied by the plume.

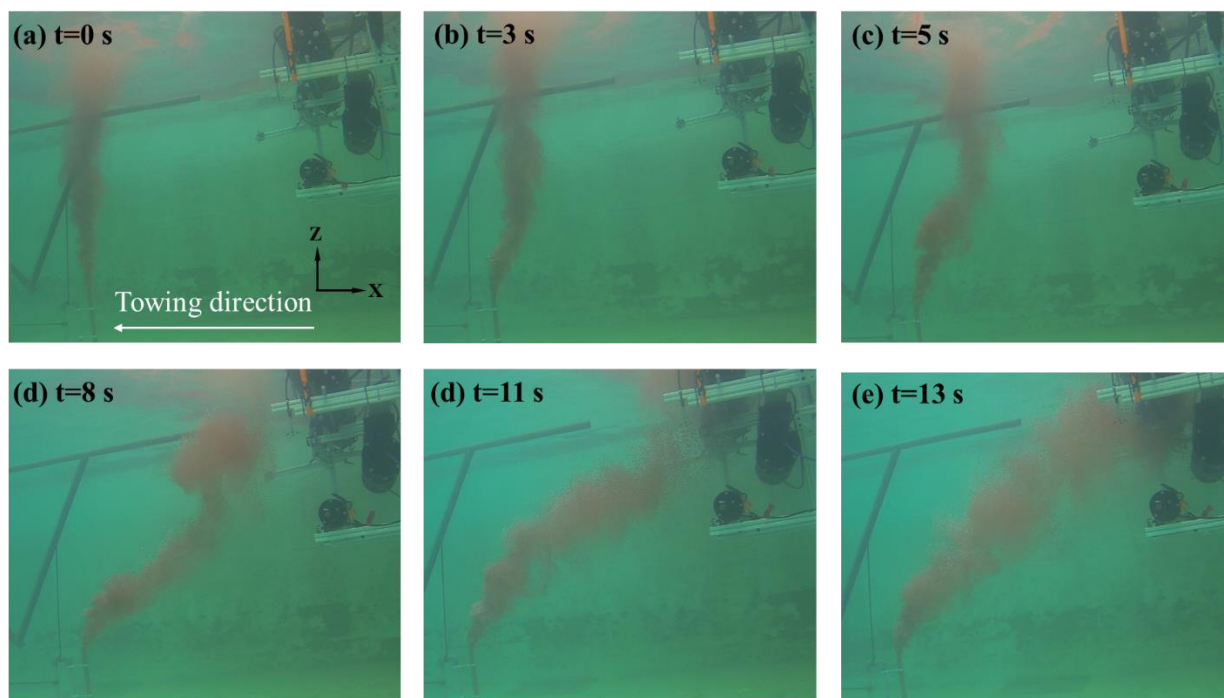
124 In the simulation, open-source computational fluid dynamics software, OpenFoam, was utilized.  
125 The mixture multiphase model (Manninen et al. 1996) and high-fidelity large eddy simulation (LES)  
126 turbulence model were employed to track the oil droplets in the turbulent flow field. In LES model,  
127 second order discretization schemes were used for all the equations except for the time discretization.  
128 First-order discretization scheme was used for the time ensuring a maximum Courant number of  $1.0$ .  
129 The time step size was around  $7.2 \times 10^{-5}$  s as the flow reached quasi-steady behavior.

### 130 **3 Results and Discussion**

131 Underwater oil blowouts including jets or plumes emanating into the stationary ambient  
132 environment were studied by our group (Zhao et al. 2014a, Zhao et al. 2014b, Zhao et al. 2016, Gao et  
133 al. 2017, Zhao et al. 2017a, Zhao et al. 2017b, Boufadel et al. 2018). The engineering models named  
134 VDROP and VDROP-J estimating droplet size distribution were proposed and validated against  
135 experimental measurements. The flow characteristics for a vertical jet into crossflow differs from the  
136 jet into a stationary flow based on different types of vortices including horseshoe vortices, wake vortices  
137 and counter-rotating vortex pair (Fric and Roshko 1994).

138 Figure 2 illustrates the time series of snapshots acquired during the experiments. Initially (at  $t=0$   
139 s), a vertical jet was obtained and waited for more than  $10$  s to allow the plume to reach quasi-steady

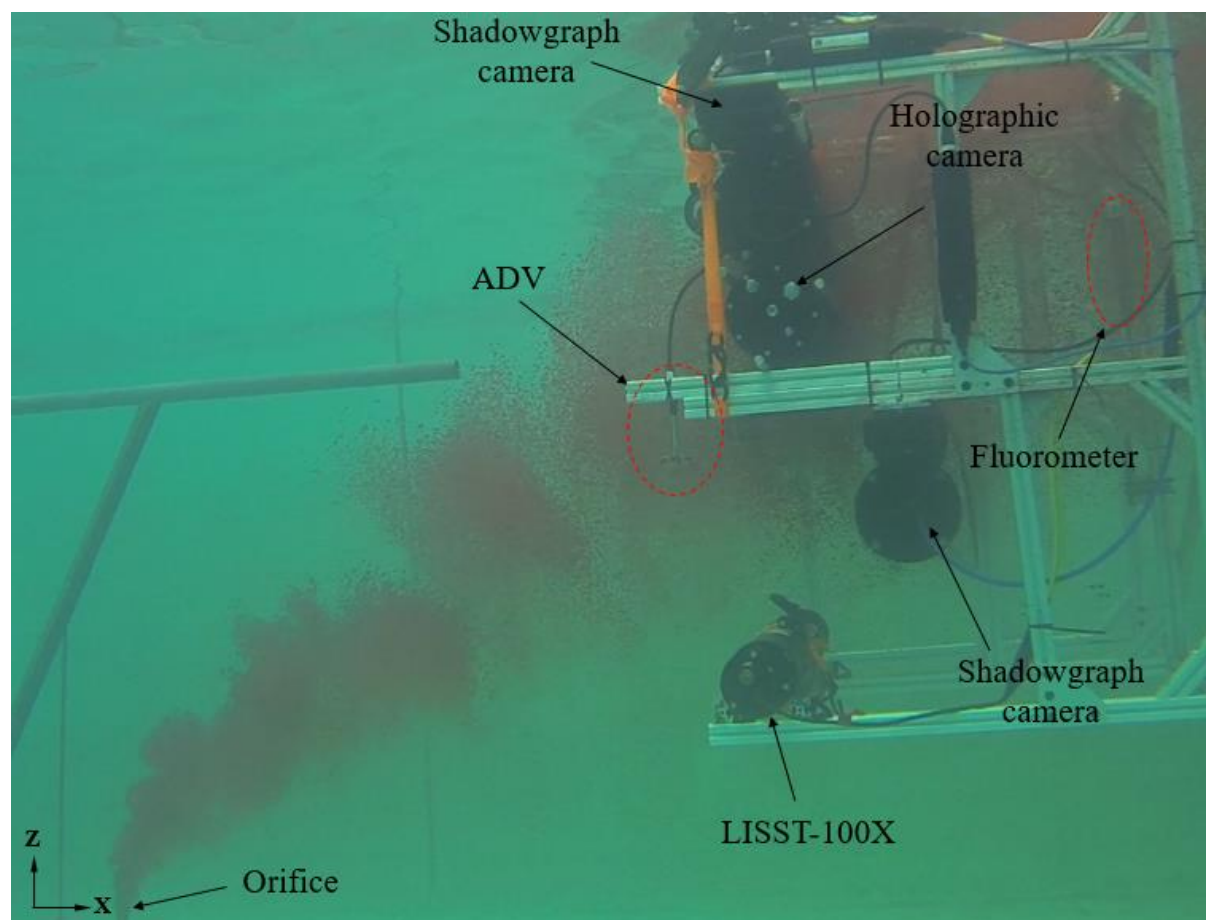
140 behavior before starting to tow. The towing direction was from right to left to mimic the crossflow from  
 141 left to right. The dynamic response of the jet to the towing can be seen in Figure 2b-d. The upstream  
 142 portion of the plume started to bend last. After almost 11 s, the plume takes its tilted shape within the  
 143 crossflow direction. The horizontal and vertical distances between the pipe orifice and the instruments  
 144 were decided attentively to allow the plume to pass through the instruments.



145  
 146 Figure 2. Snapshots from the experiment of oil jet in crossflow in consequent times. The jet was started  
 147 in the vertical direction initially. The time  $t=0$  represents the time when the towing was started.

148 The snapshot in Figure 3 reveals the trajectory of the oil plume in the crossflow. Both the upper  
 149 and lower boundaries of the plume meander in the vertical direction which is clearer in the movies. The  
 150 oil jet was almost vertical within  $2-3D$  from the orifice, then started bending in the direction of the  
 151 crossflow as its momentum decayed. The wave peaks along the leading edge of the jet, which were  
 152 induced by K-H instabilities, were clear and continuous up to  $10D$  from the orifice in the vertical  
 153 direction. The length of the waves increased in the direction of the jet path. The axial disturbances  
 154 induced by the interaction between the jet and crossflow, and the meandering behavior of the jet lead  
 155 to the column breakup which can be characterized by the disintegration of the jet core into large chunks  
 156 of oil (Behzad et al. 2016). As the far-field of the plume was focalized, the discrete large droplets were  
 157 observed near the upper boundary of the plume. In immiscible flows, the discrete droplets can separate

158 from the continuous flow due to buoyancy, turbulence and stratification (Socolofsky et al. 2002).  
 159 Therefore, larger droplets in the plume identify the upper boundary of the plume.



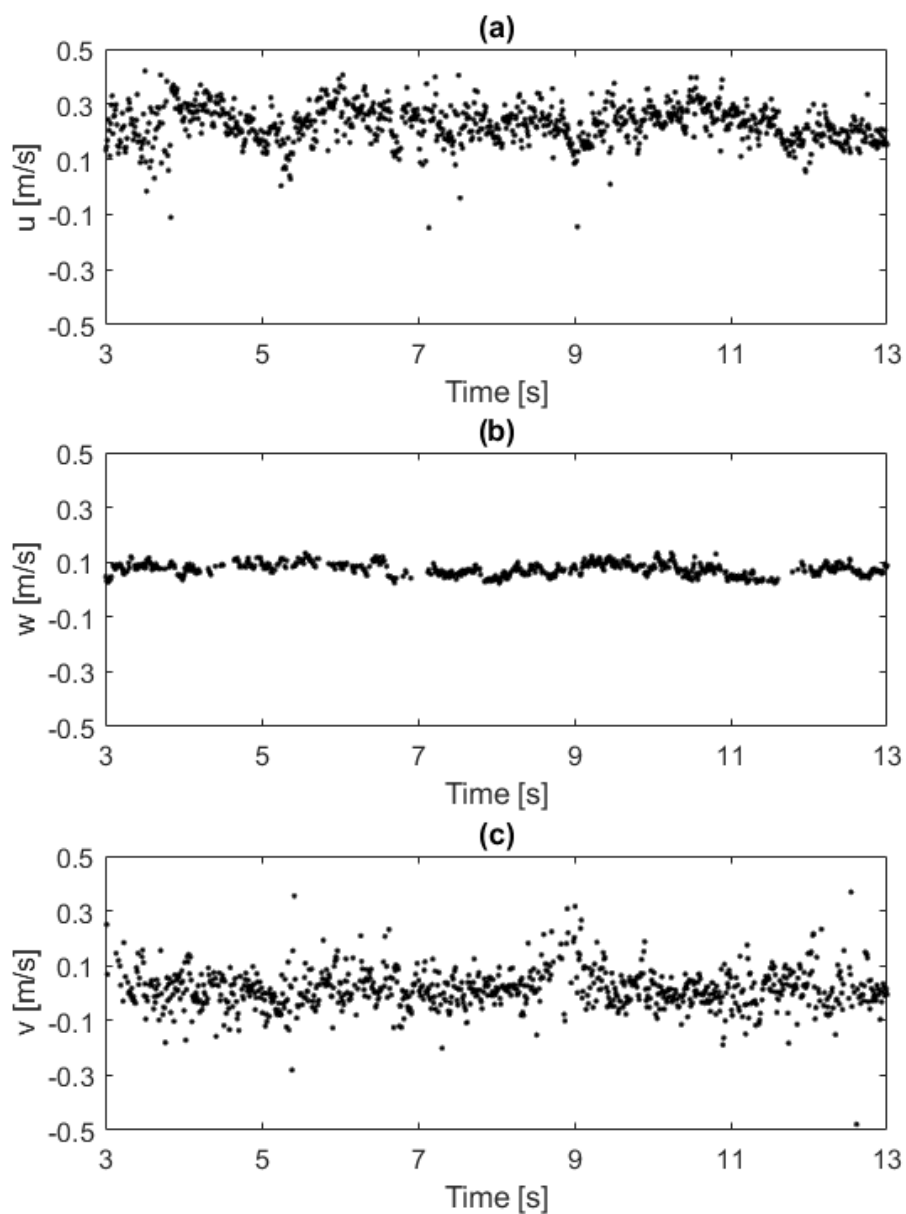
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161 Figure 3. Snapshot from the experiment of oil jet in crossflow. The oil flow rate is 40 liters/min.

162 Four ADVs (Nortek AS) were utilized to measure the instantaneous velocity components at  
 163 different locations. The accuracy of the ADV data can be evaluated based on signal to noise ratio (SNR)  
 164 and correlation coefficient. The SNR is related to the particle seeding concentration while the  
 165 correlation reveals the uncertainty in the velocity data. A higher correlation coefficient provides velocity  
 166 at higher accuracy (Mori et al. 2007). The  $SNR > 5$  is required to acquire the mean velocity while the  
 167  $SNR > 15$  is a necessity for instant velocity data (Nortek 1998, McLelland and Nicholas 2000).  
 168 McLelland and Nicholas (2000) suggested a correlation coefficient of 60% or higher for accurate  
 169 velocity output. Figure 4 illustrates the time series of velocity components acquired from the ADV-3  
 170 labeled in Figure 1. The velocity data having  $SNR < 15$  and correlation coefficient smaller than 60%  
 171 were excluded. The ADV-3 measures the velocity at  $(x,y,z)=(1.55 \text{ m}, 0.34 \text{ m}, 1.29 \text{ m})$  based on the



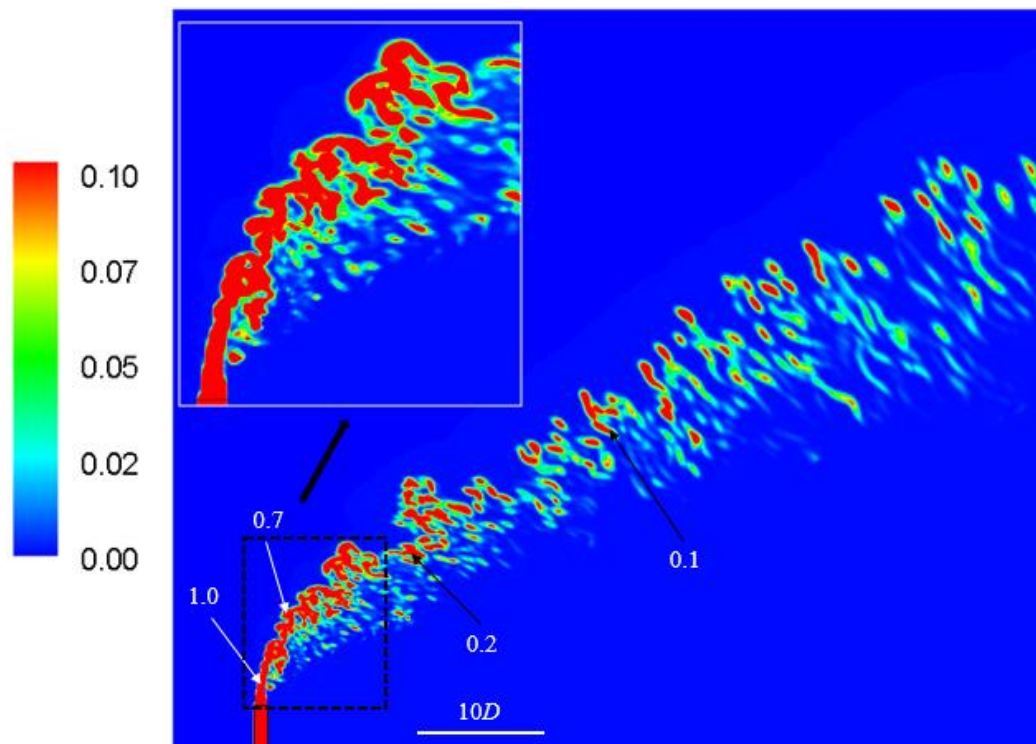
172 origin (0,0,0) located at the center of the pipe orifice. The crossflow is in the  $x$ -direction while the oil  
173 jet is in the  $z$ -direction, see Figure 3. The towing speed was 0.27 m/s and the horizontal velocity in  
174 Figure 4a oscillates around  $\sim 0.3$  m/s. The unsteady deviation from the mean velocity can be related to  
175 the meandering behavior of the plume, see Figure 3. The amplitude of the vertical velocity fluctuations  
176 was measured to be smaller than that of horizontal velocity. The vertical velocity was measured to be  
177 around 0.1 m/s. The vertical velocity for the jet in crossflow has three components: (1) jet momentum  
178 in vertical direction, (2) rise (slip) velocity of droplets and (3) flow induced by counter rotating vortex  
179 pair (Murphy et al. 2016). The velocity in lateral direction oscillates around 0 m/s between -0.1 m/s and  
180 0.1 m/s.



181

182 Figure 4. Time series of velocity in different directions measured by the ADV numbered 3 in Figure 1.  
 183 The velocities  $u$ ,  $v$  and  $w$  represent the velocities in horizontal, lateral and vertical directions. The ADV-  
 184 3 measures velocity at  $(x,y,z)=(1.55\text{ m}, 0.34\text{ m}, 1.29\text{ m})$  based on the origin  $(0,0,0)$  located at the center  
 185 of pipe orifice. The figure was obtained from Daskiran et al. (2020b).

186 Figure 5 illustrates the numerical predictions of instantaneous oil holdup along the symmetry plane.  
 187 The near-field of the jet was magnified and placed top-left. The trajectory and meandering behavior of  
 188 the jet, wavy pattern along the leading edge of the jet and the column breakup of the plume observed in  
 189 the experiments captured well within the simulations. Initially, the jet moved in the vertical direction  
 190 and bent in the horizontal direction slightly up to  $3D$  from the orifice. The jet completed its bending  
 191 smoothly within  $10D$  from the orifice and raised almost with a constant slope. The wavy behavior  
 192 induced by K-H instabilities was observed near the leading edge of the jet and lasted nearly  $10D$ .  
 193 Following the wavy pattern, column break was observed due to axial disturbances (Behzad et al. 2016).  
 194 In the near field of the jet, the surface breakup induced by azimuthal instabilities created ligaments and  
 195 droplets along the trailing edge of the jet (Behzad et al. 2016). The dilution of oil along the jet path was  
 196 also quantified. At  $30D$  away from the pipe orifice, the oil holdup decreased to 0.1. As the plume diluted  
 197 more, the distance between the oil pockets was increased. The oil holdup was estimated to be higher  
 198 near the upper boundary of the plume.



199

200 Figure 5. Instantaneous contours of oil holdup at  $t=10$  s. The oil is discharged into the streaming water  
201 from a pipe with a diameter of 2.5 cm. The water stream is in  $+x$  direction while the oil jet is in  $+z$   
202 direction.

#### 203 **4 Conclusions**

204 Large scale experiments of oil jet in crossflow were performed at Ohmsett wave tank facility and  
205 simulated herein through the computational fluid dynamics approach. Several instruments including  
206 ADVs, LISSTs, shadowgraph cameras, holographic camera, and fluorometers were used in the  
207 experiments. In contrast to most of the prior studies, a larger orifice with a diameter of one-inch was  
208 used in this work which is crucial to adjust correlations that were proposed to estimate median droplet  
209 size ( $d_{50}$ ). Li et al. (2017) argued that using the same correlations for a relatively larger pipe diameter  
210 might result in a droplet diameter larger than the maximum stable droplet size. Using both LISST and  
211 shadowgraph cameras in this work aids in capturing the whole droplet size range from  $1.0 \mu\text{m}$  to cm  
212 scale.

213 The trajectory and meandering behavior of the oil jet, wavy pattern along the leading edge of the  
214 jet and column breakup observed in the experiments were captured well with the numerical simulation.  
215 The surface breakup near-field of the jet induced by azimuthal instabilities created ligaments and  
216 droplets downstream of the trailing edge of the jet. Larger oil droplets escaped from the entrained water  
217 and they were observed near the upper boundary of the plume since the larger droplets have higher  
218 buoyancy (Boufadel et al. 2007) to overcome the turbulent mixing while the smaller droplets were  
219 suspended and tracked the water flow.

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227 Supercomputer Center at UC San Diego and the *Bridges* system at the Pittsburgh Supercomputing  
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